Boosting performance in hydraulic systems with alternative coatings

Taking a holistic approach to sealing solutions in hydraulic systems using chrome alternatives helps engineers achieve optimum function.

By Beth Figliulo and Michael Cook, Trelleborg Sealing Solutions

In 2017, the Registration, Evaluation, Authorization and Restriction of Chemicals (REACH) directive placed restrictions on the use of any coating or plating using hexavalent chromium (Cr6) to protect humans and the environment from potentially harmful chemicals.

As a result, alternative coatings are becoming more common for coating piston rods in hydraulic systems. However, most surface-finish guidance from seal suppliers is still based on 2D finish characteristics for use with chromium layers. This can significantly affect performance and system life in hydraulic systems because the lubrication within these systems is fundamentally affected by the surface finish of the mating surfaces.

Instead, a 3D approach is needed, taking into consideration the intersection of seals, fluid and mating surface finish. Tribology — the science and engineering of interacting surfaces in relative motion, including the study and application of the principles of friction, lubrication and wear — should be optimized. This article explains how to fully optimize the tribology effect within hydraulic systems using chrome alternatives.

Understanding roughness and surface finishes

Whether single- or double-acting, a hydraulic cylinder converts energy stored in the hydraulic fluid into a force used to move the cylinder in a linear direction. The cylinder barrel contains a piston connected to a piston rod. Coatings are usually applied to the piston rod to improve its friction characteristics, maximizing performance and minimizing wear. Those coatings are finished to roughness recommendations from seal suppliers so that seals achieve optimum function.

Surface roughness describes deviations in the direction of the normal vector of a surface from its ideal form. Large deviations mean a rough surface; small deviations indicate a smooth surface.

The most commonly used roughness parameter is the Ra, which is the arithmetic average of all absolute values of the profile height deviations from the mean — the average of a set of individual measurements of a surface’s peaks and valleys.

Two other 2D measures are also commonly used: Rz and Rz1max. Rz is the average measurement of the vertical distance from the highest peak to the
lowest valley in five sample lengths. The Rz1max takes the largest of the five Rzi values from the five sampling lengths over the total measured length.

The issue with continuing to use these 2D measures is that alternative coatings have a fundamentally different topography.

**Alternative coatings**

The two main types of alternative processes used to coat piston rods are high velocity oxygen fuel (HVOF) and laser metal deposition (LMD). The former uses the combustion of a gas (e.g., oxygen) and a liquid fuel (e.g., kerosene) to spray a layer of tungsten carbide, chromium carbide or nickel onto the rod in a thickness of about 0.1 mm to 0.3 mm.

The latter uses a laser to generate a molten bath on the existing surface into which one or more metal powders is sprayed. Typical materials sprayed onto the surface include nickel, steel and/or carbide blends. These materials coat the rod in thicknesses between 0.2 mm and 4 mm and are often softer than HVOF or chrome layers.

The typography of both HVOF and LMD differ significantly from chrome in their Ra, Rz and Rmax values, having significantly fewer 2D profile height deviations. This means that to ensure proper film formation, other surface finish parameters must be considered.

### Table 1: Extended approach to the description of mating surface.

<table>
<thead>
<tr>
<th></th>
<th>Ra</th>
<th>Rz</th>
<th>Rpk</th>
<th>Rvk</th>
<th>Rmr*</th>
</tr>
</thead>
<tbody>
<tr>
<td>Min</td>
<td>Max</td>
<td>Min</td>
<td>Max</td>
<td>Min</td>
<td>Max</td>
</tr>
<tr>
<td>Rods, hard chrome</td>
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<td>0.3</td>
<td>0.6</td>
<td>2.0</td>
<td>0.05</td>
</tr>
<tr>
<td>Mineral Oil / HFC PTFE Compound</td>
<td>0.05</td>
<td>0.4</td>
<td>0.6</td>
<td>2.0</td>
<td>0.05</td>
</tr>
<tr>
<td>Rods, Welded and Sprayed Overlays ≥45 HRC</td>
<td>0.05</td>
<td>0.3</td>
<td>0.6</td>
<td>2.0</td>
<td>0.05</td>
</tr>
<tr>
<td>Rods, HVOF, Ceramic Welded and Sprayed Overlays ≥45 HRC</td>
<td>0.05</td>
<td>0.2</td>
<td>0.3</td>
<td>1.5</td>
<td>0.03</td>
</tr>
<tr>
<td>Bores, Cylinders honed/grinded</td>
<td>0.05</td>
<td>0.4</td>
<td>0.6</td>
<td>2.5</td>
<td>0.05</td>
</tr>
<tr>
<td>Min</td>
<td>Max</td>
<td>Min</td>
<td>Max</td>
<td>Min</td>
<td>Max</td>
</tr>
<tr>
<td>Mineral Oil / HFC PTFE Compound, Polyurethane (TPU)</td>
<td>0.05</td>
<td>0.4</td>
<td>0.6</td>
<td>2.0</td>
<td>0.05</td>
</tr>
<tr>
<td>Bores, Single accumulators honed/grinded</td>
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<td>0.6</td>
<td>2.0</td>
<td>0.05</td>
</tr>
<tr>
<td>Bores, Single accumulators roller burnished</td>
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<td>0.3</td>
<td>0.6</td>
<td>2.0</td>
<td>0.05</td>
</tr>
</tbody>
</table>

*Rmr determined at cut depth C = 0.25 x Rz, relative to reference line Cref = 5% for soft coatings and chrome Cref = 2% for nitrided and hardened "Cref = 0% for ceramic coating.

**Understanding tribology**

In hydraulic systems, tribology refers to proper tuning of the seal, the hydraulic fluid and the mating surface. To optimize a hydraulic system using either HVOF or LMD, the suitability of the surface should be based on 3D parameters: the material ratio curve (Rmr), the reduced peak height (Rpk), the reduced valley depth (Rvk) and the core region (Rk).

This creates a roughness profile based on the average depth of the grooves projecting from the core area into the material, the average height of the peaks protruding from the material and the depth of the roughness kernel profile. This is a much more accurate way of describing mating surfaces than Ra alone or even Ra with Rz and Rmax.

However, to truly optimize system tribology, one must also take into consideration coatings, piston manufacturing methods and piston type. This requires evaluating the run-in behavior of the seal and mating surface by measuring the surface topography with different depths of cut, with a cutting reference of 0%, 2% and 5%. This will allow the different wear behaviors of each surface coating to be considered.

**Holistic analysis**

By differentiating in terms of coating, manufacturing process and distinction of piston rod or piston tube, an extended approach for describing mating surfaces can be created. This includes the standard measurements of Ra and Rz and extends them to include Rpk and Rvk along with Rmr at various percentage cutting references based on the coating. Additionally, upper and lower limits can be set to properly describe the full topography of the various surfaces.

Extensive dynamic tests using a variety of sealing loads and materials in both rod and piston configurations were conducted to provide the data for Table 1. The leakage of the system and the wear behaviors of both the seals and mating surface ultimately determined the limits for each parameter.

**Lubricant-optimized hydraulic sealing systems**

Although surface topography plays a critical role in system life, as emphasized in the previous section, it is not the only factor that can affect how long equipment performs at optimum levels. An increased understanding of surface finish and tribology can be combined with lubrication management technology to realize a fully optimized hydraulic sealing system.

Lubrication management involves using a tandem sealing arrangement where
the primary seal has a rounded contact surface that enables an ideal lubrication film to be formed that also extends to the secondary seal.

This phenomenon has been proven through side-by-side testing on laser-clad piston rods with identical surface topography based on the previously highlighted recommendations. The results showed significant reductions in surface smoothing for the system employing lubrication management.

Importantly, the system with lubrication management experienced reduced friction and up to four times less seal wear on both the primary and secondary seals.

This outcome effectively demonstrates how lubrication management can further extend system life. However, it is important to understand this can only be achieved with a surface-finish recommendation that allows the effective tuning of the tribology of the hydraulic system.

**Conclusion**

Detailed surface finish measurements have shown that recommendations for chrome-plated surfaces do not completely characterize the necessary parameters for surfaces using alternative coatings. With an extended approach to measurements and analysis, numerous tests have allowed for the benchmarking of proper surface finish recommendations for the full spectrum of surface coatings and hardware configurations.

Furthermore, testing on these tuned surfaces has shown that the current performance limits of sealing systems can be considerably extended by adding lubrication management technology, thus producing an even higher degree of robustness than tribologically assessed mating surfaces alone.

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